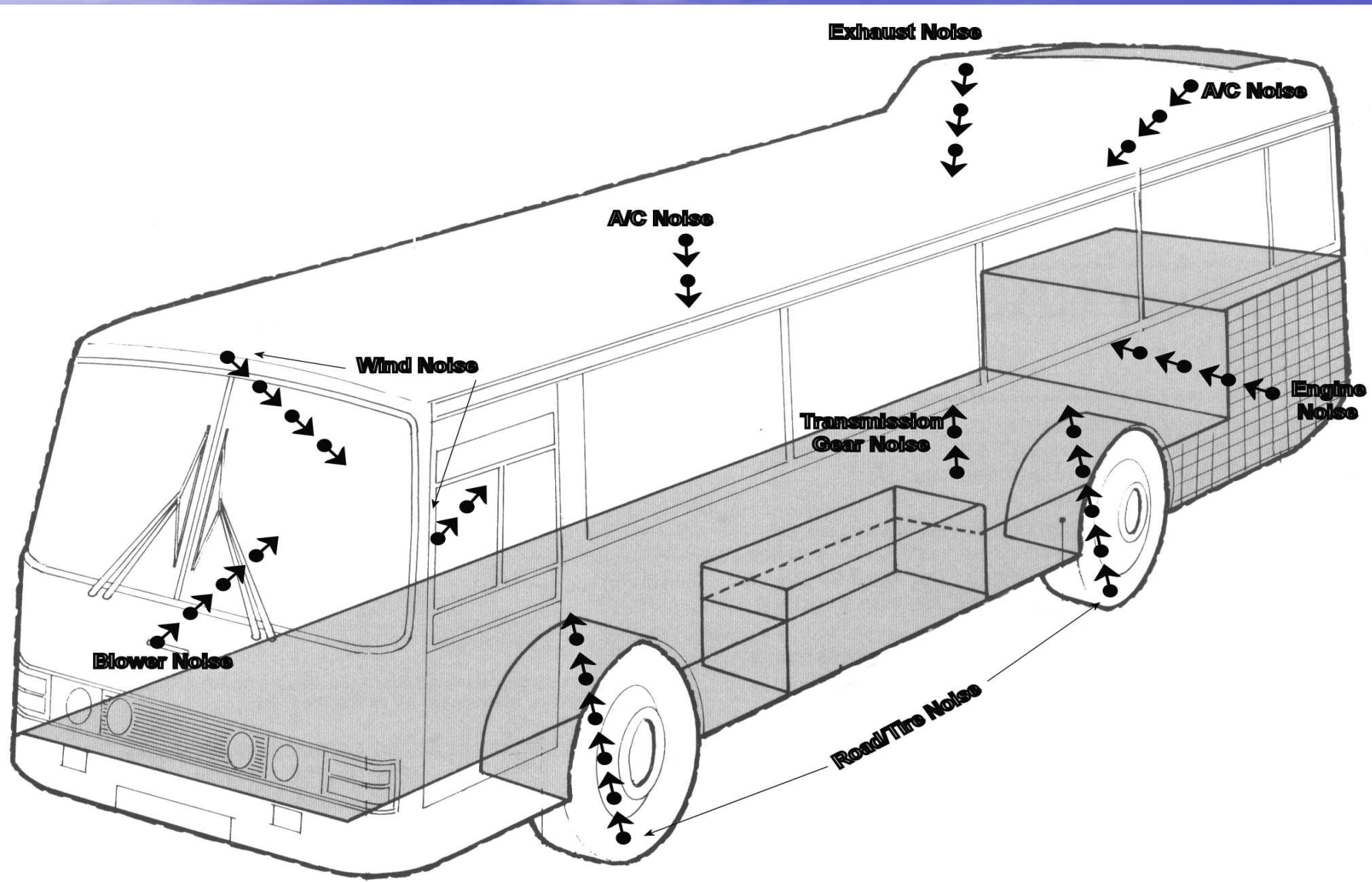


Lightweight, non-flammable noise-control methodology for transit vehicles

Peter Jackson
Technical Director
American Acoustical Products

Transit Vehicle Noise



Acoustic Methodology

Design for silence

- Treat Source of Noise
- Treat Noise Paths
- Treat Receiver Environment

Acoustic Materials

- Absorbers

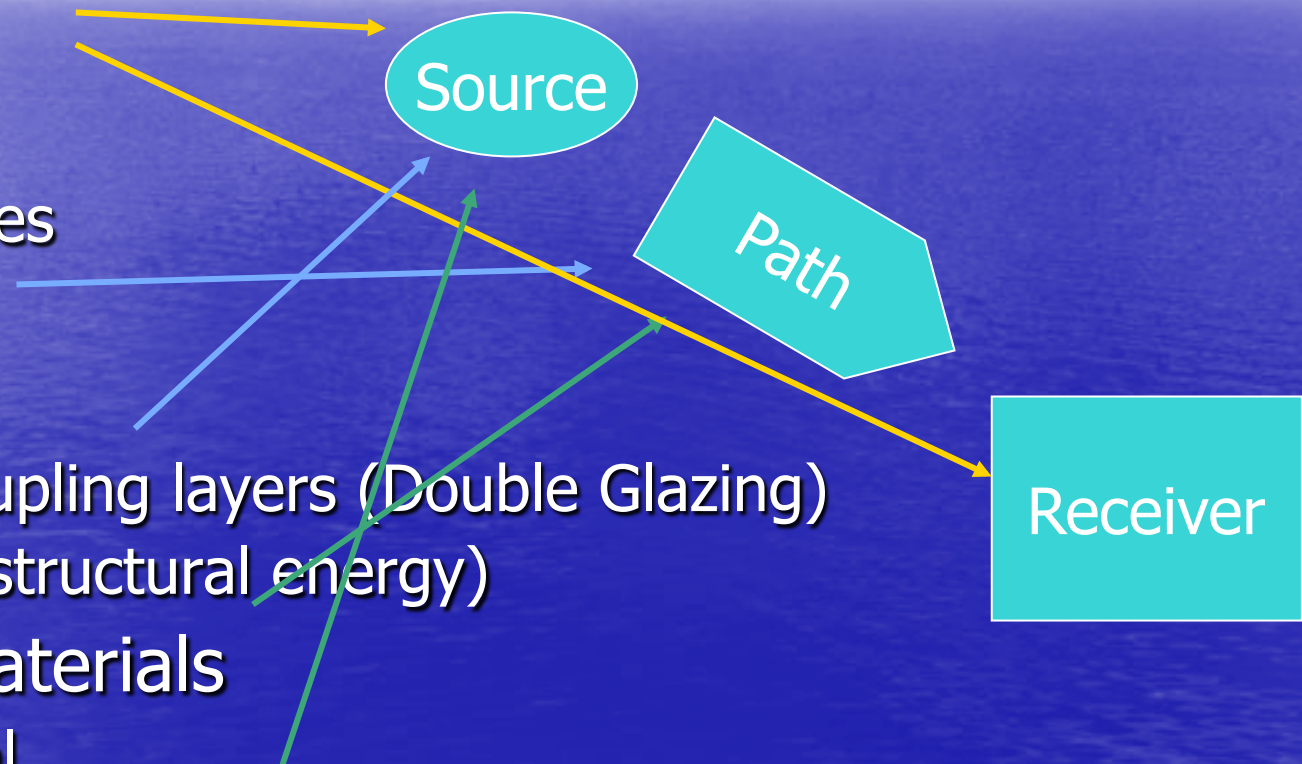
- Basic
- With cavities

- Barriers

- Simple
- With decoupling layers (Double Glazing)
- Isolators (structural energy)

- Damping Materials

- Extensional
- Constrained Layer



Treat Source of Noise

- Usually Most Cost Effective in Long Run
 - One Source Many Paths
- Vibration Isolation
- Acoustical Shielding
- Sound Absorption

Noise Sources

- Engine Intake and Exhaust
- Engine Cooling Fan
- Tire or Rail Noise
- Compressors and Blower Noise
- HVAC Noise
- Wind Noise

Engine Intake and Exhaust

- Engine Firing Frequency
- Reactive Muffler
- Tuned Side Ports for Specific Frequencies
- Intake Air from Outside Engine Compartment
- Exhaust Gas Away from Sensitive Direction

Engine Cooling Fan

- Decouple from Engine
- Swept Blades (Avoid Pure Tones)
- Avoid Struts close to blades (Avoid Turbulence)
- Control Fan Speed (N)
 - $SPL \propto N^5 \text{ or } 6$
 - $Q \text{ (volume)} \propto N \text{ and Size}$
 - $P \text{ (Pressure)} \propto N^2$



Flammability Issues with Foam Products



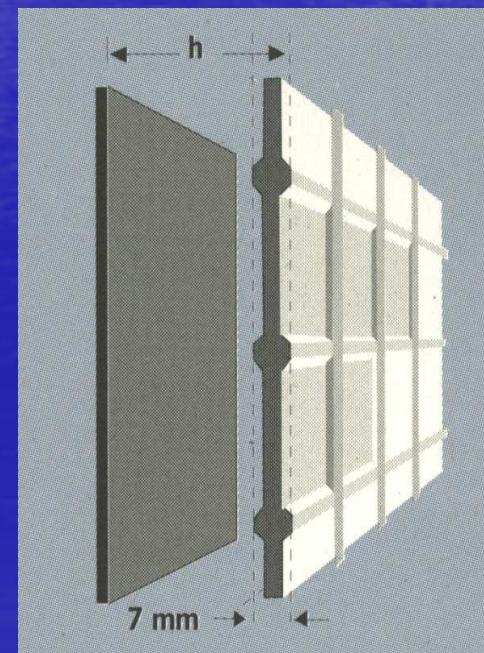
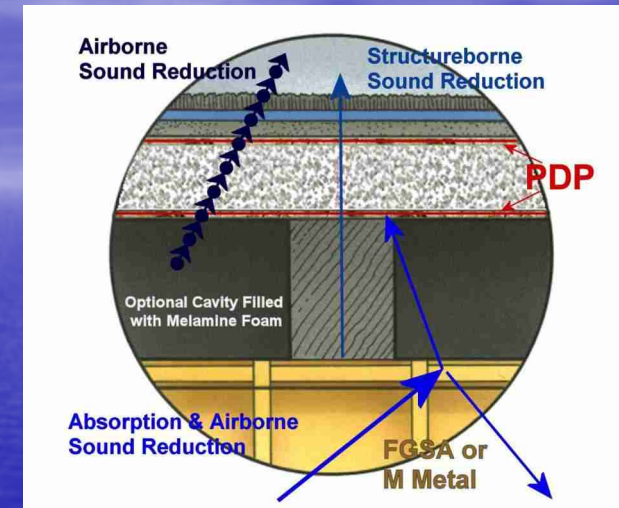
Engine Compartment Absorption

- Cover about 60% of free surfaces
- Spaced Absorber on common wall/floor areas for enhanced Transmission Loss



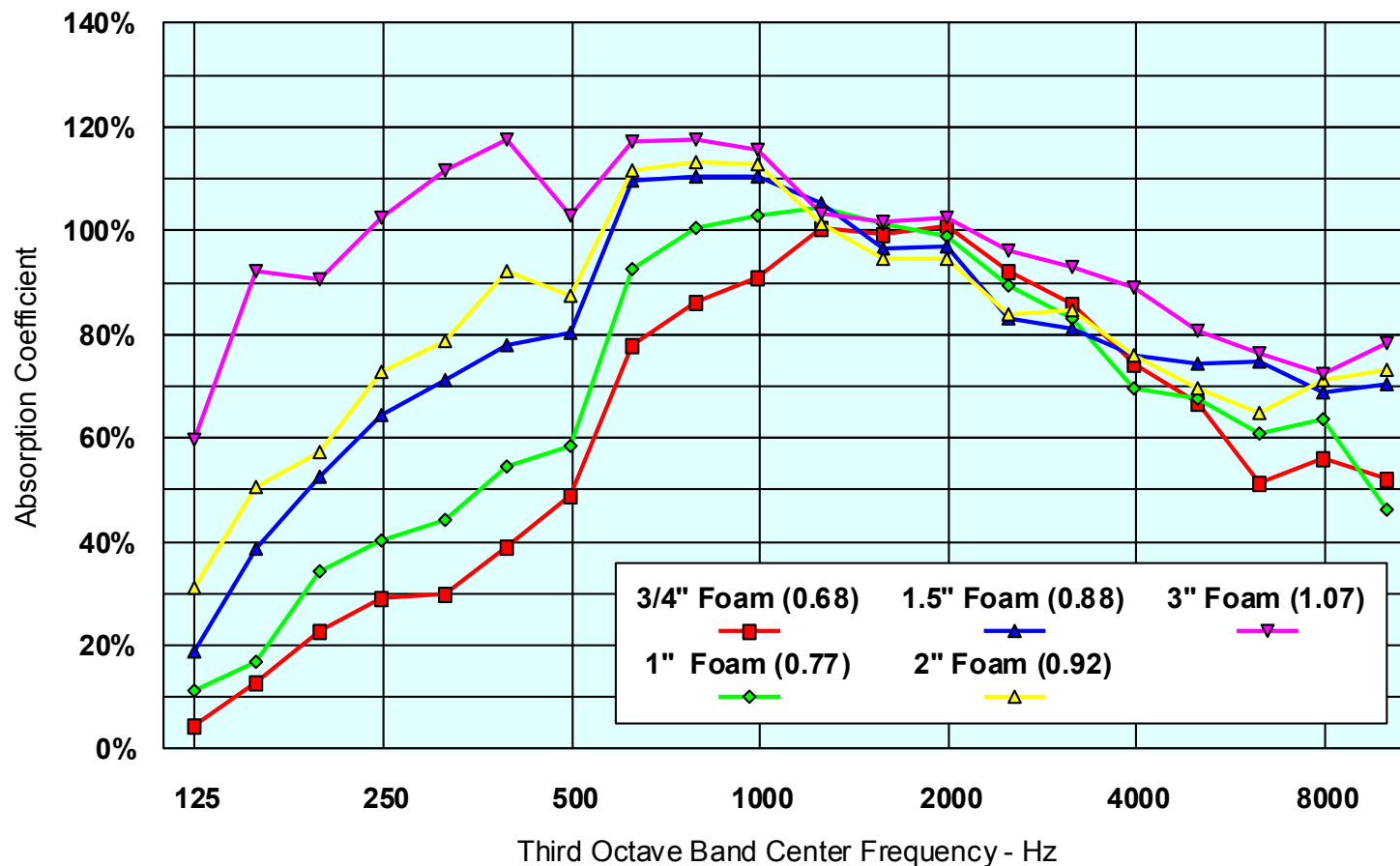
Spaced Absorbers

- Enhanced Low Frequency
- Decoupled Impedance
- FGSA – Film facing
- Millennium Metal
- Foam in Cavity –Further Enhances Low Frequency



Effect of Spacing

Millennium Metal with Melamine Foam
Figures in (Brackets) AAC - Average Absorption Coefficient
Results normalized to ASTM C 423-99



Wheel and Tire Noise

- Noise Mechanics
 - Air Displacement
 - Tire Tread Block Impact
- Regular Pattern Creates Tones
- Tread Designed to Lay Down not Impact
- Rubber Compound to be High Resilience

Blower Noise

- Driver Noise Requirement [75dB(A)]
- Fan Speed Reduction
 - $\text{SPL} \propto (\text{Fan Speed})^5 \text{ or } 6$
- Smoother Airflow
- Larger Fans(= Slower for same duty)
- Heated Glass Instead of demisting Blowers

HVAC Noise

- Decouple Compressor from Engine
- Control Fan Speed
 - $SPL \propto N^5 \text{ or } 6$
 - $Q \propto N$
 - $P \propto N^2$
- In Duct or Plenum Absorption Materials
- Move Air Return to Floor Level (85dB(A) Limit in Cabin)

Wind Noise

- Design
 - Aerodynamic Mirrors
 - Avoid Sharp curves
 - Add spoilers to deflect flow from weak areas
- Window Seals
- Door Seals - double or triple
- Double Glazing and or laminated Safety Glass

Acoustic Methodology

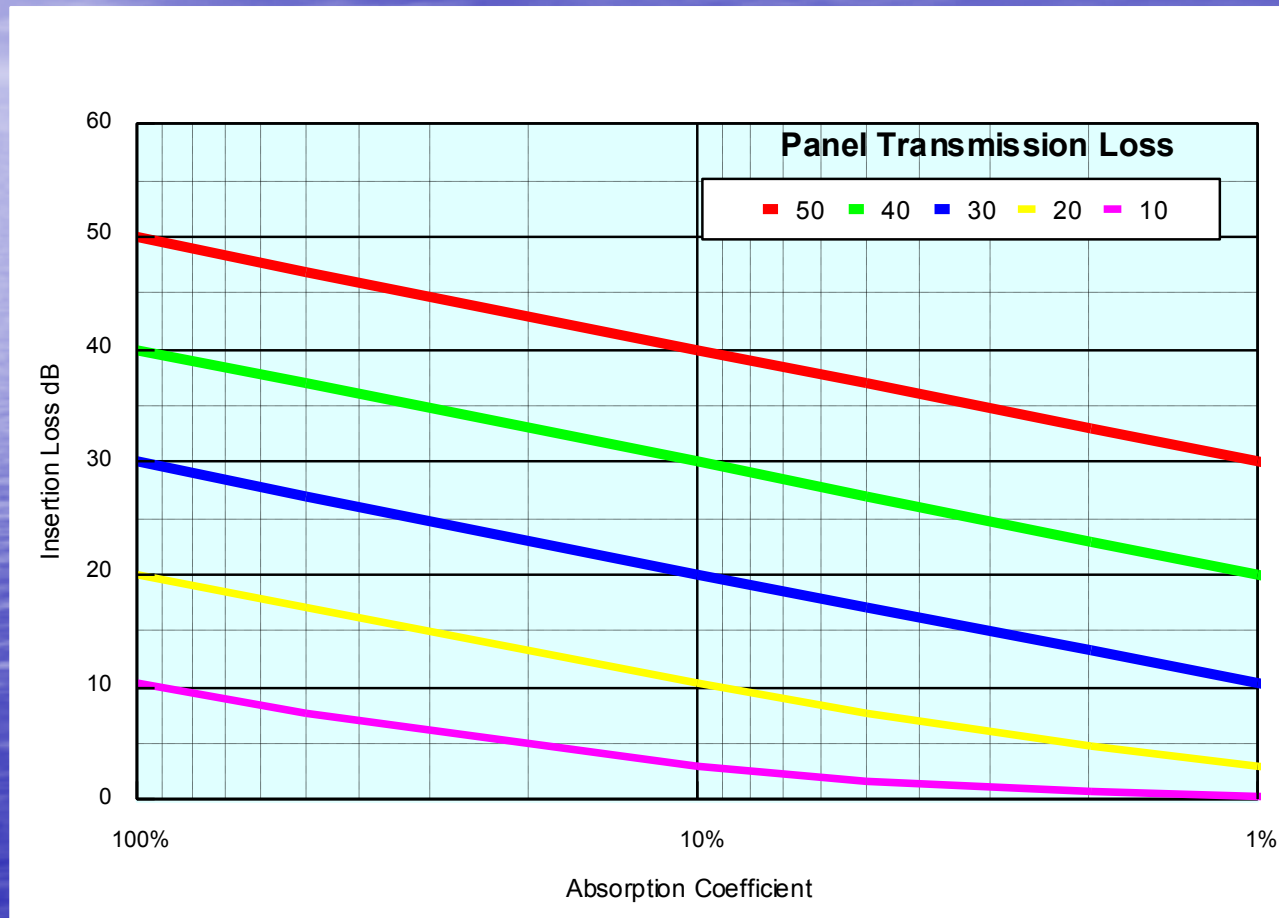
Design for silence

- Treat Source of Noise
- **Treat Noise Paths**
- Treat Receiver Environment

Noise Path

- Sound Absorption
- Close off all holes and pass-through areas
- Mass Controlled Attenuation
- Bending Wave reduction
- Vibration Isolation
- Vibration Attenuation

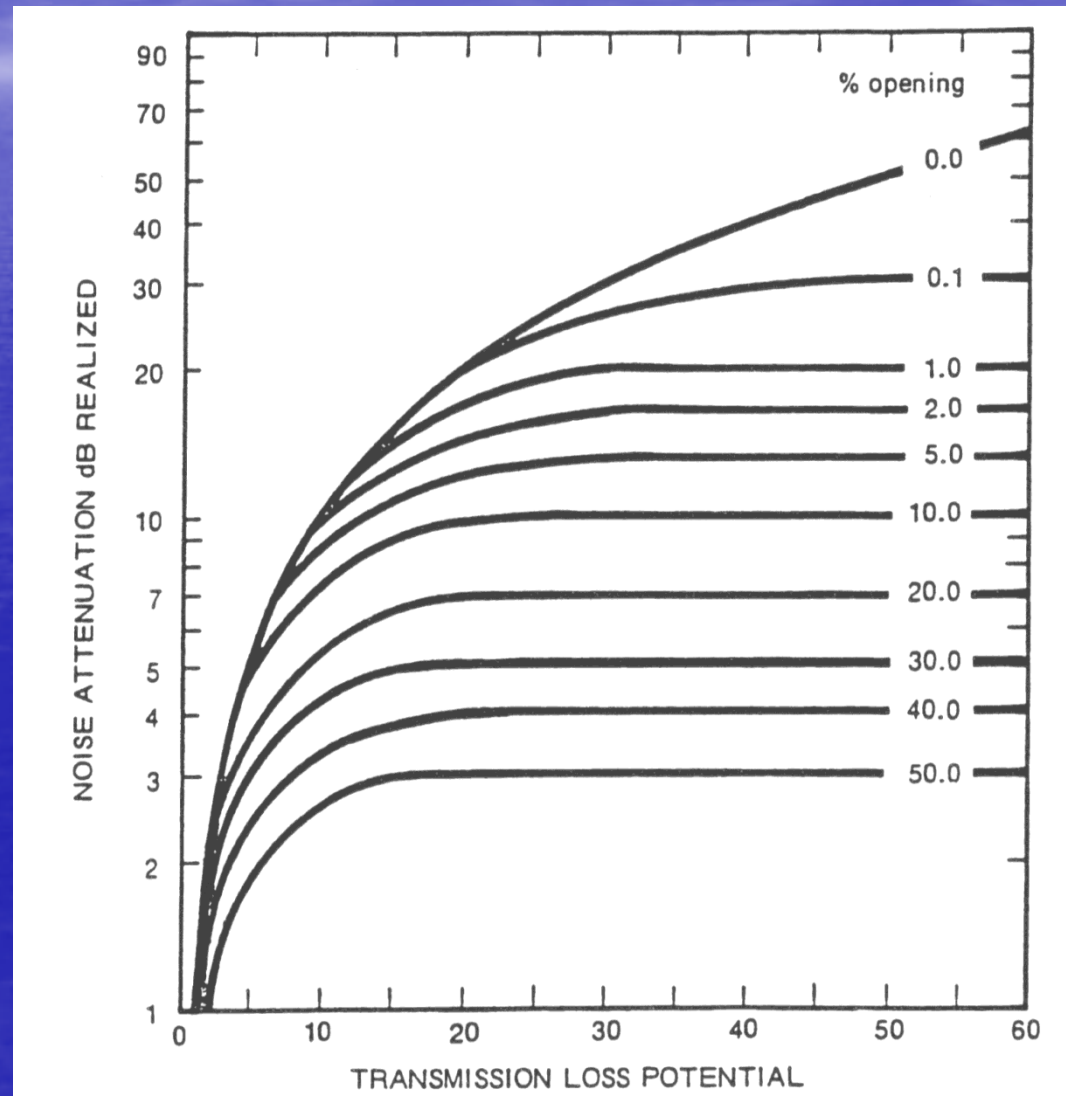
Effect of Absorption on Insertion Loss



$$IL = TL + 10 \cdot \log(\alpha + \tau) \quad \text{where } \tau = 1/(10^{TL/10})$$

Effect of Holes on Sound Transmission

- STL is limited by holes
 - 0.1% 30dB
 - 1% 20dB
 - 10% 10dB

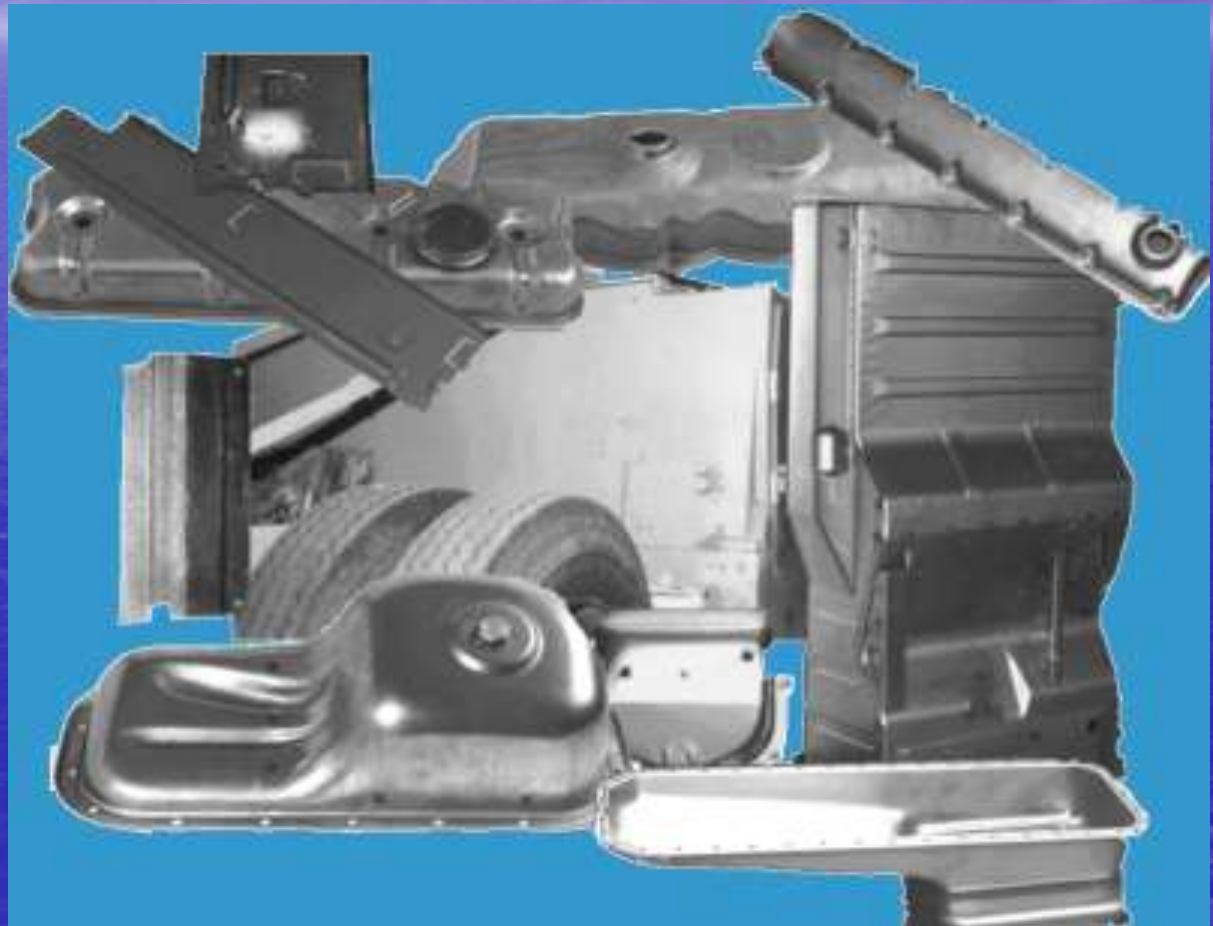


Damping

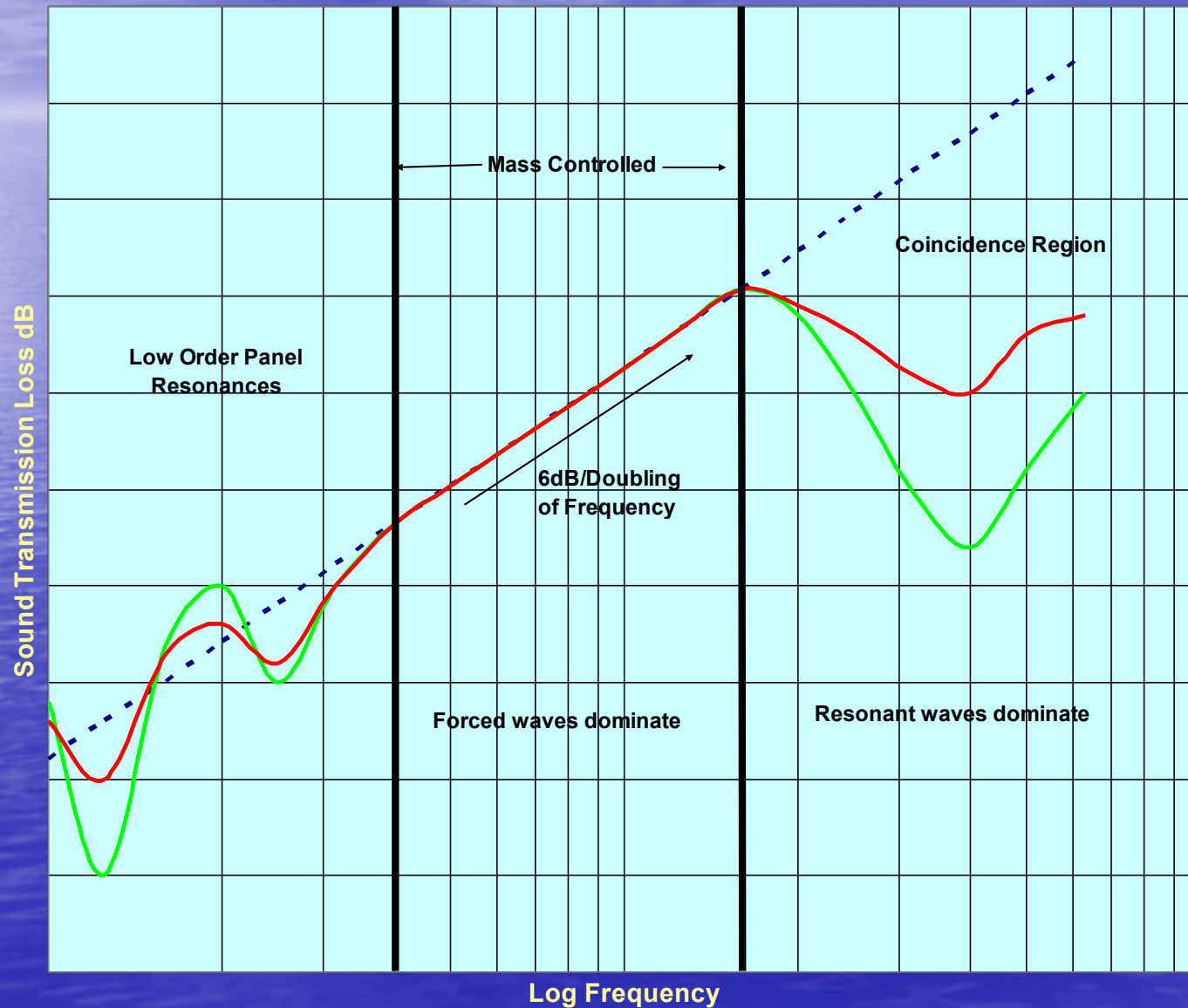
- Extensional Damping
 - Moduli Mismatched
 - Excessive Amounts Required
 - Material Exposed – Flammability Durability etc
- Constrained Layer Damping
 - Viscoelastic Material at or Near Neutral Axis
 - Constraining Layer Matches Bending Stiffness of Structure

Noiseless Metal

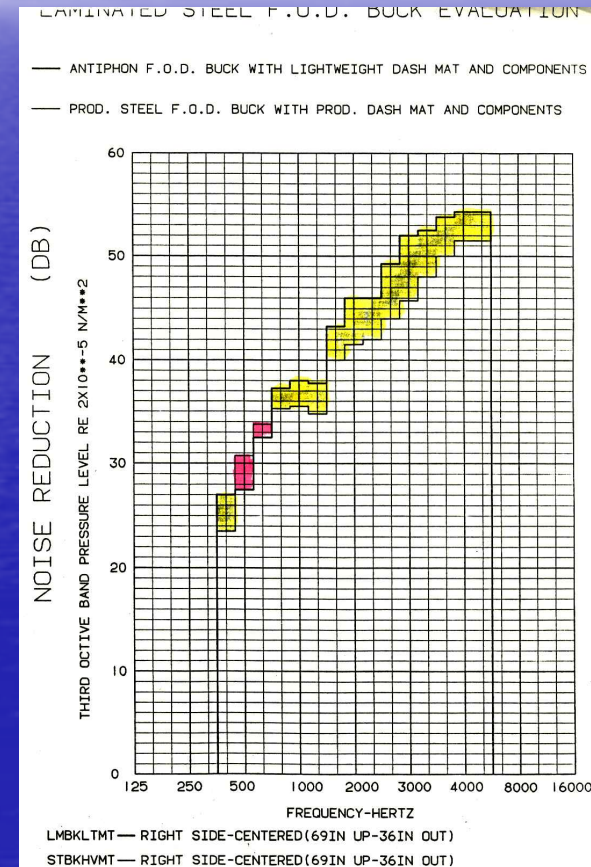
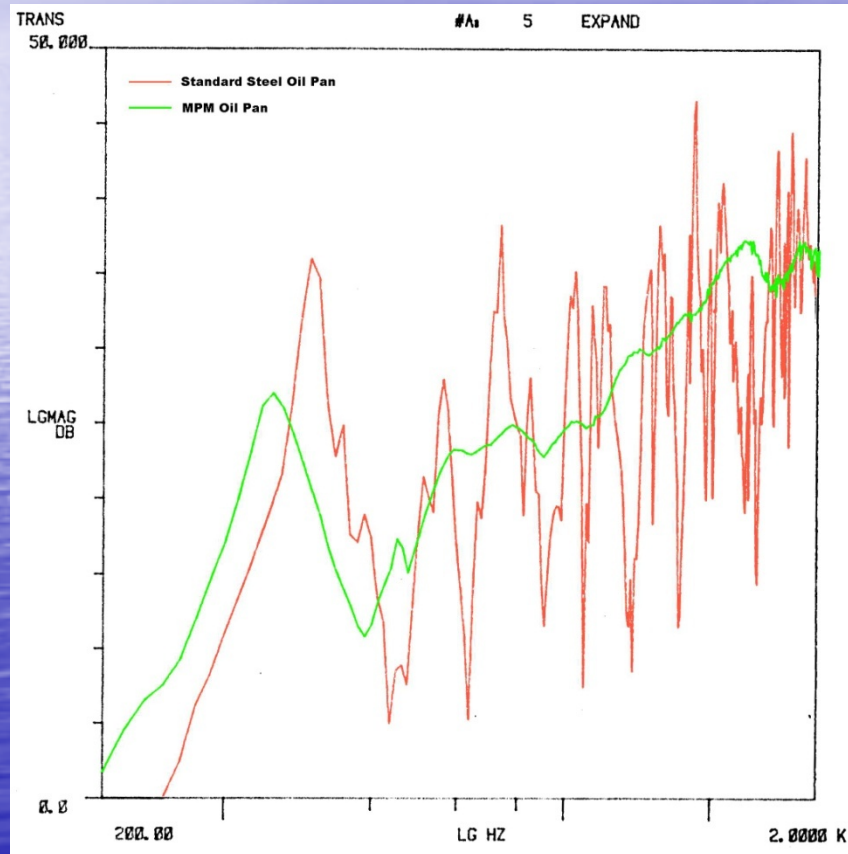
- Formable
- Paintable
- Weldable



STL: The Reason



Noiseless Metal Acoustics



Plywood Floor Transmission Loss

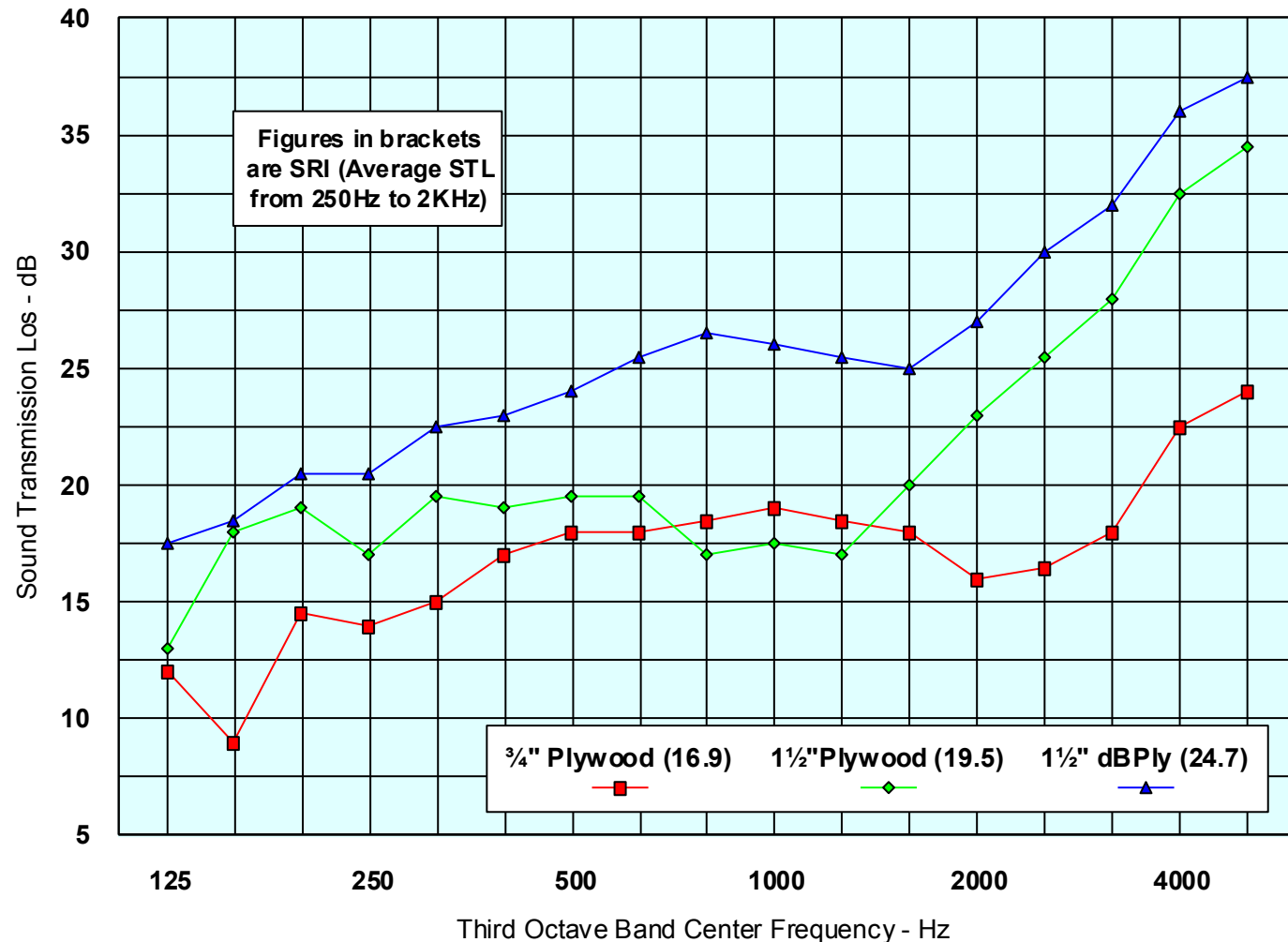
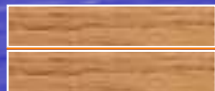
3/4" Plywood



1 1/2" Plywood



1 1/2" dBply



PDP

- Noiseless Fiberglass
- Encapsulated to Protect Inner Layer
- Apply to relatively Flat areas Only
- Improves STL
- Absorbs Structural Noise Energy

Foam Cored Panels

Before

10mm(3/8") Fiberglass



60mm (2
3/8")
Divinacell

10mm(3/8") Fiberglass

After

2 x 10mm(3/8") Fiberglass



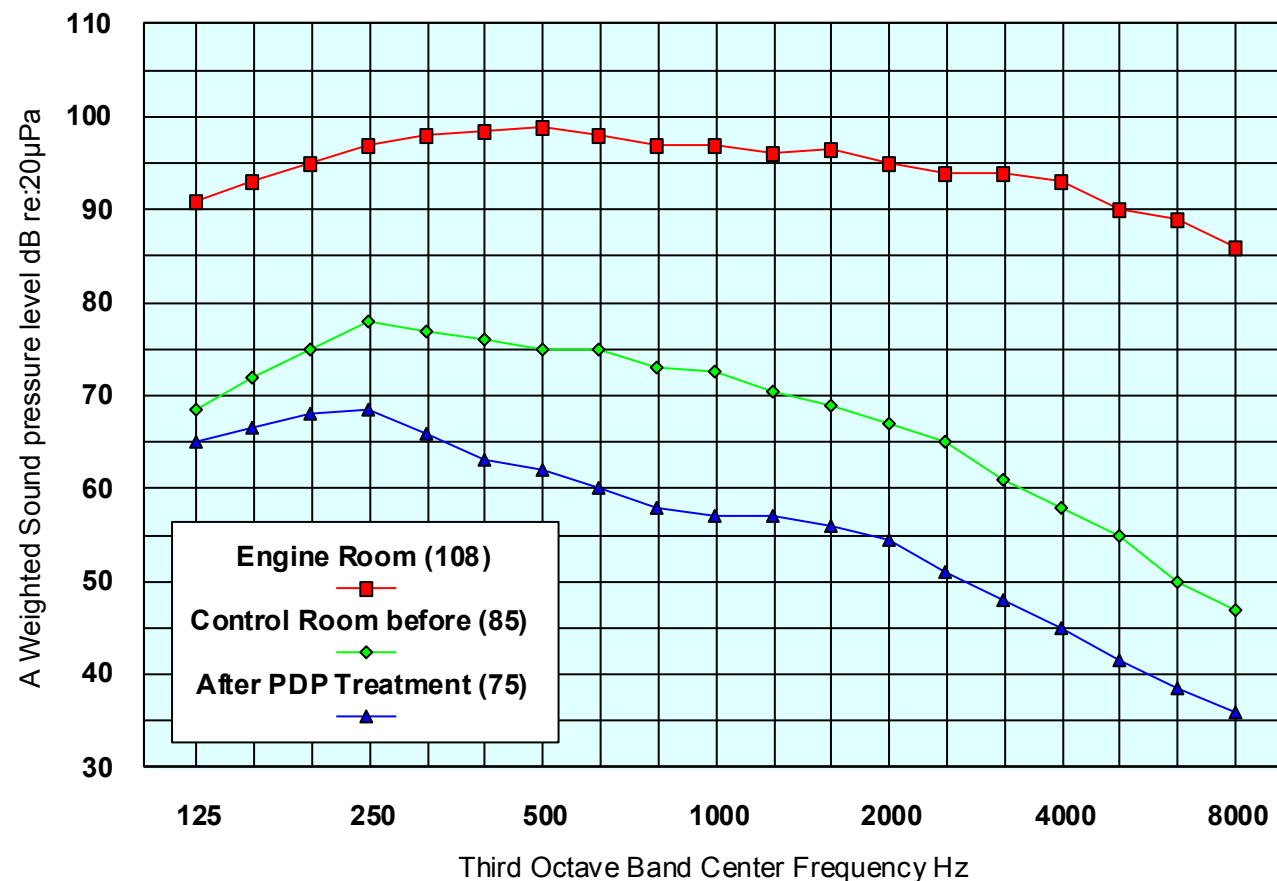
← PDP

60mm (2
3/8")
Divinacell

10mm(3/8") Fiberglass

Improvement by Damping Bulkhead with PDP

Figures in (brackets) A Weighted Sound pressure Level



Acoustic Methodology

Design for silence

- Treat Source of Noise
- Treat Noise Paths
- **Treat Receiver Environment**

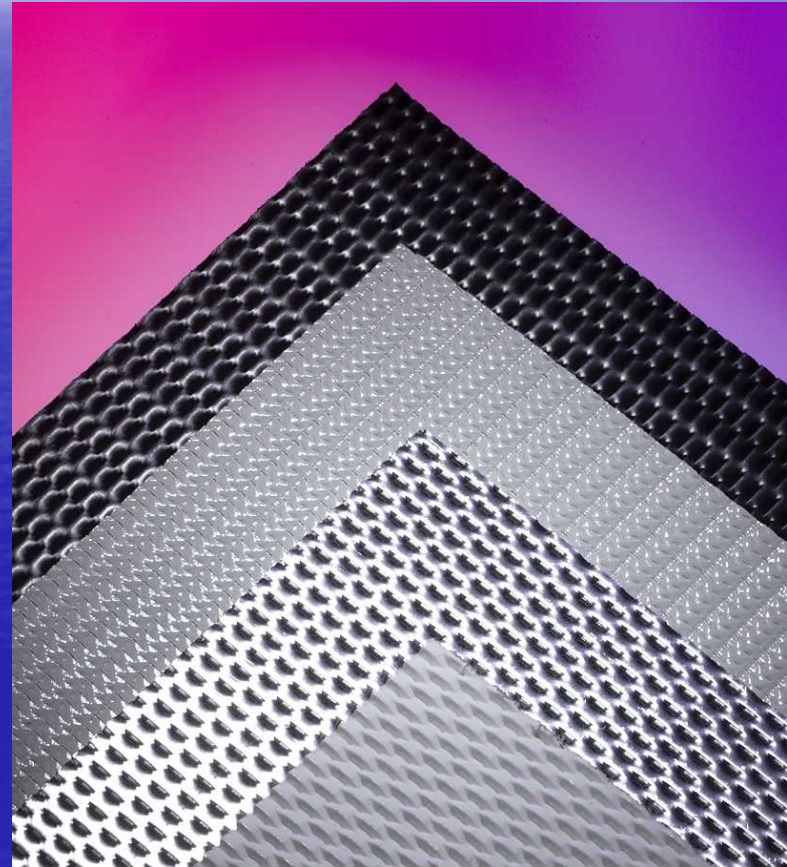
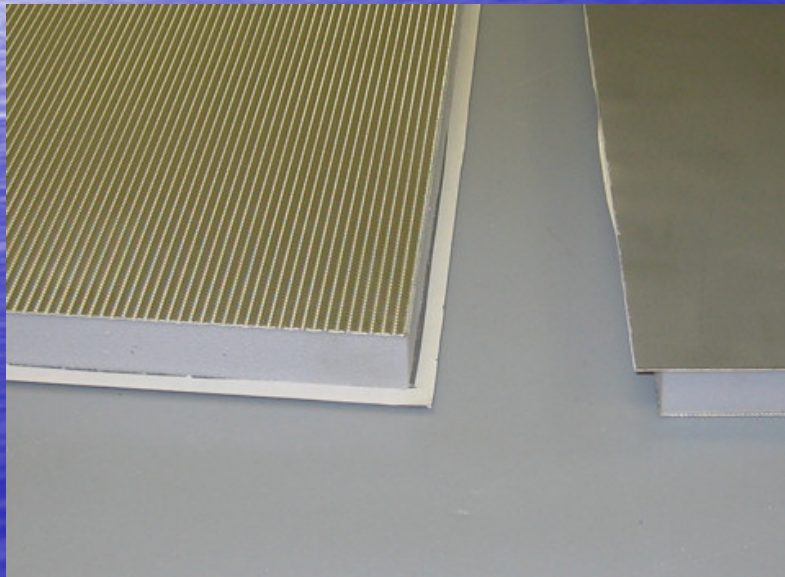
Absorption



- Close to Driver – Additional Absorption and Screening
- Absorption panels in walls and ceiling of main cabin
- Absorptive Seating – Melamine foam for smoke and fire suppression

Millennium Metal

- Tough Durable Facing
- Optimized Acoustics
- Structural Panel



Interior Absorption

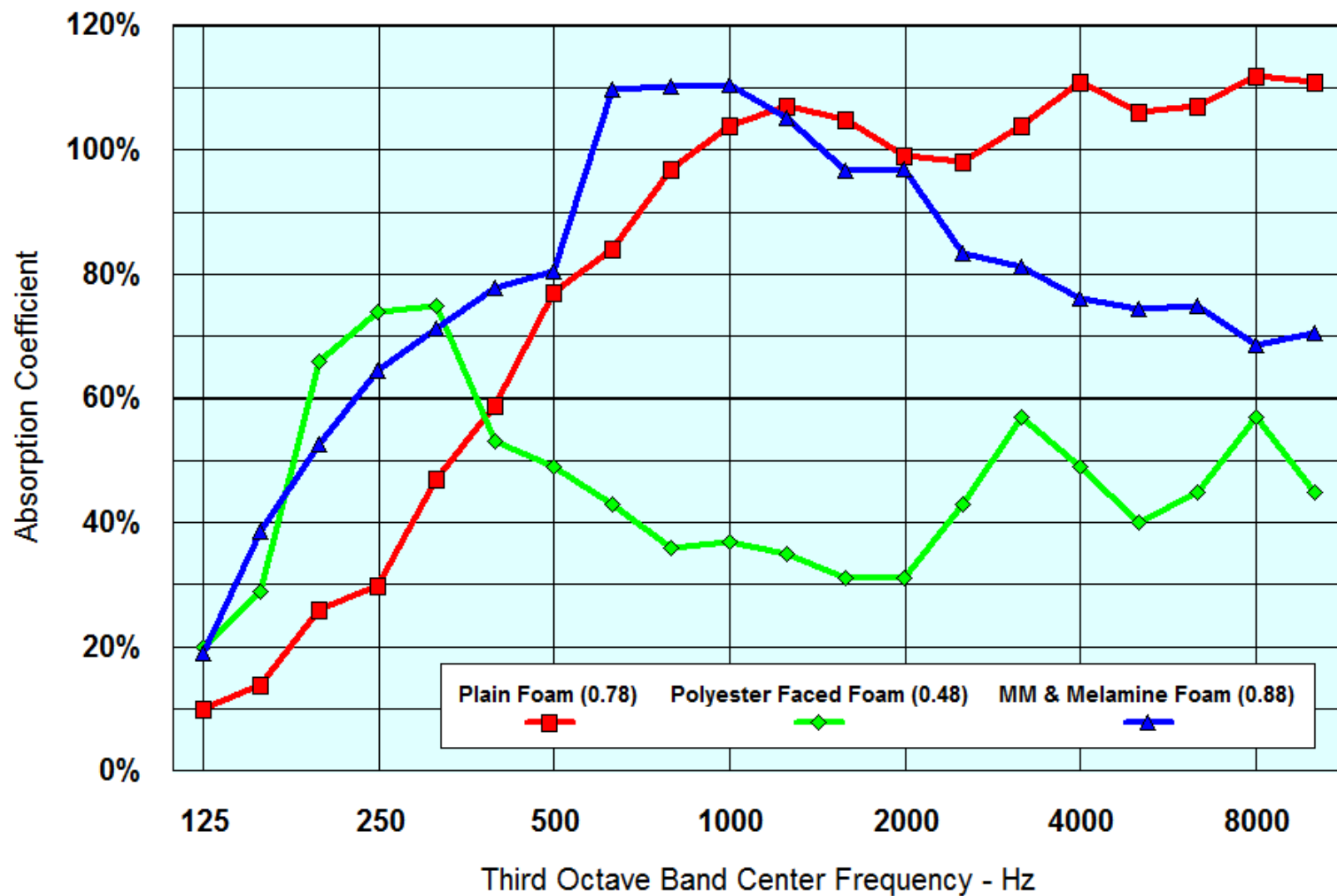
- Powder Coated Millennium Metal with Melamine foam Thermal Insulation
- Safe Durable Fire resistant decorative



Comparison of Absorption of 1.5" (38mm) Absorption

Figures in (Brackets) AAC - Average Absorption Coefficient

Results normalized to ASTM C 423-99



Summary

- Closing Holes and leaks is Job #1
- Reduce Fan speeds
- Millennium Metal & Melamine in Engine compartment = BIC for Noise and Flammability
- Constrained Layer Damping = BIC STL for Optimum Weight Noiseless Metal, dBPLy & PDP in Bulkhead and Wheel Wells
- Constrained layer damping (dBPLy) for floor
- Millennium Metal & Melamine in Roof of Cabin