



## Synergy of the main technologies supported downsized engine

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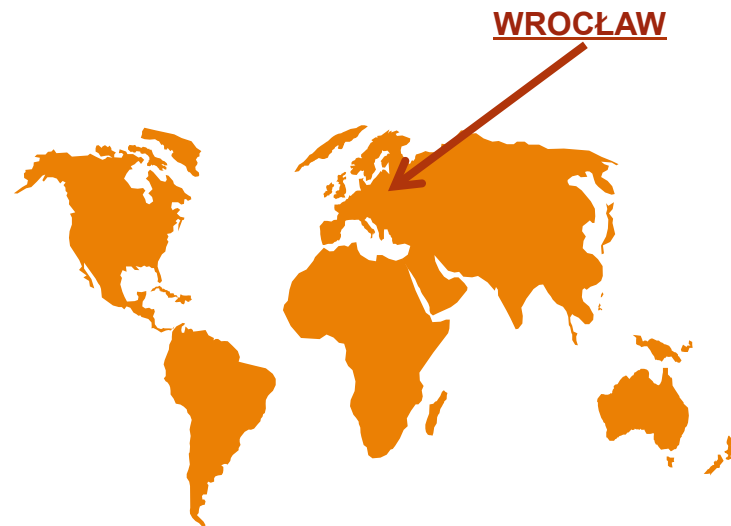


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## Agenda

- What is downsizing ?
- Why does downsizing use?
- How can downsizing realize ?
- Geometry and supported technologies.
- Baseline engine vs. downsized mix engine.

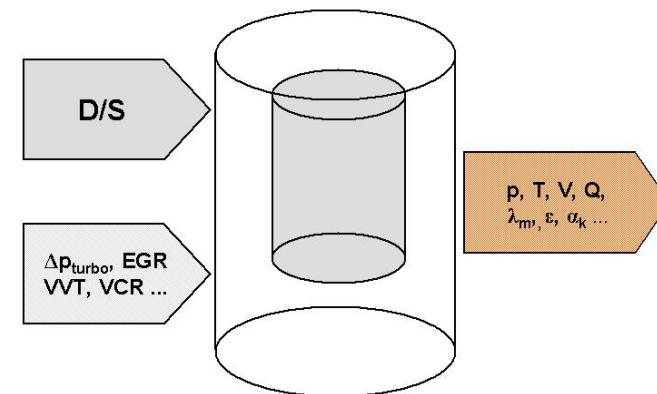


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In global terms, transport has a significant share in the production of  $\text{CO}_2$ , approaching the level of 30%, as the combustion engine still remains the dominant source of drive power, and in principle, it is responsible for producing carbon dioxide. That indicates the presence of a real threat to the existence of transport or the necessity of its significant reduction and it has an impact on countries' economies and politics. For these reasons automotive research centers and academic institutions worldwide organize numerous projects aimed at improvements in the design of combustion engines.

Downsizing is one of the latest trends in development. It means reducing the engine displacement while maintaining or increasing its performance. The downsizing concept incorporates a reduction in fuel consumption, and hence, the decreased emission of carbon dioxide to the atmosphere.



## What ?



Consideration example due to author's research

If a reduction in CO<sub>2</sub> at the level of 20%, resulting from the implementation of the concept of combustion engine downsizing (which corresponds to a reduction in displacement by about 30%), is adopted for all vehicles operated on the routes Wrocław - Dresden - Prague, during one day, the ecological effect connected with the decreased emission of carbon dioxide to the atmosphere will amount up to 79 Mkg.

On an annual basis the ecological effect of downsizing will total almost 20 Million kg.



# Why ?



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Downsizing can be done by changing:

- in stroke ( $A = S_d/S$ ),
- in diameter ( $B = D_d/D$ )
- both parameters - as mix.

Downsizing ratio:  $W_d = 1 - (V_d/V) \Rightarrow W_d = 1 - A \cdot B^2$

where:

$V_d$  - swept volume of downsized engine

$V$  - swept volume of baseline engine

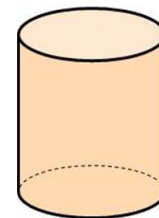
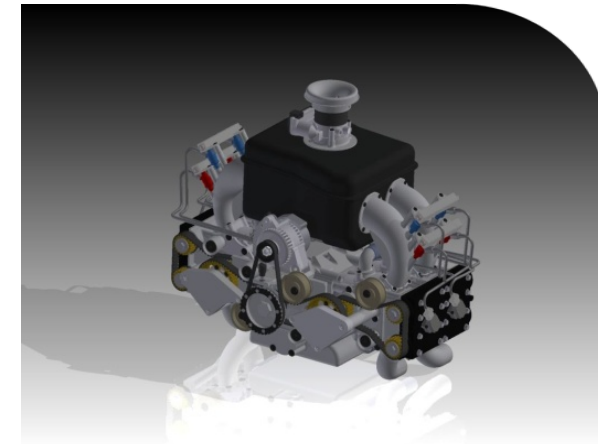
$S_d$  - downsized stroke,

$S$  - baseline stroke

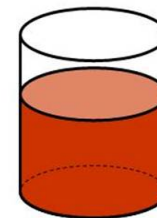
$D_d$  - downsized bore

$D$  - baseline bore

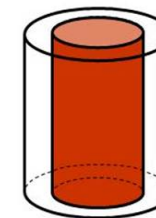
		Changing B for cylinder diameter						
		1,00	0,96	0,90	0,85	0,80	0,75	0,70
Changing A for piston stroke	1,00	0,00	0,08	0,19	0,28	0,36	0,44	0,51
	0,95	0,05	0,12	0,23	0,31	0,39	0,47	0,53
	0,90	0,10	0,17	0,27	0,35	0,42	0,49	0,56
	0,85	0,15	0,22	0,31	0,39	0,46	0,52	0,58
	0,80	0,20	0,26	0,35	0,42	0,49	0,55	0,61
	0,75	0,25	0,31	0,39	0,46	0,52	0,58	0,63
	0,70	0,30	0,35	0,43	0,49	0,55	0,61	0,66



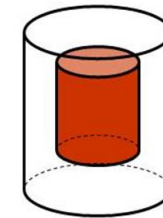
$W_d=0$



$W_d=1-A$



$W_d=1-B^2$



$W_d=1-AB$

The same results in swept volume reduction

# How ?



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Data for baseline engine			
name	symbol	Value	unit
Geometry – some parameters			
Cylinder bore	D	84,80	mm
Piston stroke	S	88,00	mm
Ratio of cylinder bore to piston stroke	$R_{BL}$	0,96	
Engine swept volume	$V_S$	1988,04	cm <sup>3</sup>
Clearance volume	$V_C$	212,62	cm <sup>3</sup>
Working cycle – some factors			
Coefficient of pressure drop during filling stroke	$C_p$	0,95	
Coefficient of filling	$\eta_v$	0,87	
Compression ratio	$\epsilon$	9,35	
Relative air/fuel ratio	$\lambda$	0,92	
Maximum theoretical pressure of cycle	$p_{max}$	7,54	MPa
Mean friction pressure	$p_f$	0,20	MPa
Average piston speed	$S_p$	17,60	m/s
Mean effective pressure	BMEP	0,94	MPa
Mechanical efficiency	$\eta_m$	82,27	%
Total efficiency	$\eta_o$	28,37	%
Specific fuel consumption	$g_e$	295,09	g/kWh
Engine power at 6000 rpm	$P_e$	93,16	kW



BASELINE ENGINE DOWNSIZED by 30%												
geometry changing only												
SYMBOL	UNIT	INITIAL VALUE	RESULTS									
D	mm	84,80	84,80	0%	81,30	-4%	76,50	-10%	74,50	-12%	71,10	-16%
S	mm	88,00	61,80	-30%	67,30	-24%	76,00	-14%	80,10	-9%	88,00	0%
VW <sub>d</sub>	cm <sup>3</sup>	1988,04	1396,14	- 30%	1397,48	-30%	1397,29	-30%	1396,67	-30%	1397,56	-30%
$\eta_v$	-	0,87	0,87	0%	0,87	0%	0,87	0%	0,87	0%	0,87	0%
p <sub>max</sub>	MPa	7,54	7,54	0%	7,54	0%	7,54	0%	7,54	0%	7,54	0%
BMEP	MPa	0,94	0,99	+5%	0,98	+4%	0,96	+3%	0,95	+2%	0,94	0%
g <sub>e</sub>	g/kWh	295,10	279,76	-5%	282,84	-4%	287,87	-2%	290,30	-2%	295,10	0%
P <sub>e</sub>	kW	93,16	69,01	- 26%	68,32	-27%	67,12	-28%	66,53	-29%	65,49	-30%
CO <sub>2</sub>	kg/h	86,87	61,01	-30%	61,06	-30%	61,06	-30%	61,03	-30%	61,07	-30%

 Less than initial value

 More than initial value

D- cylinder diameter  
S- piston stroke  
V - swept volume,  
W<sub>d</sub>- downsizing ratio,  
 $\eta_v$  - volumetric efficiency,  
p<sub>max</sub> - max pressure in chamber,  
BMEP - brake mean effective pressure,  
g<sub>e</sub> - specific fuel consumption,  
P<sub>e</sub> - effective power,  
CO<sub>2</sub> - carbon monoxide



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DOWNSIZED ENGINE by 30% (for D=76,50mm and S=76,00mm)												
Turbo support only												
SYMBOL	UNIT	INITIAL VALUE	RESULTS									
$C_p$	-		1,00		1,25		1,50		1,75		2,00	
$\eta_v$	-	0,87	0,92	+6%	1,18	+35%	1,43	+65%	1,69	+94%	1,95	+124%
$p_{max}$	MPa	7,54	7,95	+5%	10,14	+35%	12,32	+64%	14,51	+93%	16,70	+122%
BMEP	MPa	0,96	1,03	+7%	1,40	+45%	1,76	+84%	2,13	+122%	2,50	+160%
$g_e$	g/kWh	287,87	283,54	-2%	267,73	-7%	258,50	-10%	252,45	-12%	248,17	-14%
$P_e$	kW	67,12	71,95	+7%	97,58	+45%	123,21	+84%	148,85	+122%	174,48	+160%
CO <sub>2</sub>	kg/h	61,06	64,464	+6%	82,556	+35%	100,647	+65%	118,738	+95%	136,830	+124%

Less than initial value

More than initial value

$C_p$  - coefficient of pressure drop during filling stroke,  
 $\eta_v$  - volumetric efficiency,  
 $p_{max}$  - max pressure in chamber,  
 BMEP - brake mean effective pressure,  
 $g_e$  - specific fuel consumption,  
 $P_e$  - effective power,  
 CO<sub>2</sub> - carbon monoxide





DOWNSIZED ENGINE by 30% (for D=76,50mm and S=76,00mm)												
Variable Compression Ratio support only												
SYMBOL	UNIT	INITIAL VALUE	RESULTS									
CR	-		6,00		8,0		10,00		12,00		14,00	
$V_c$	cm <sup>3</sup>	149,44	232,88	+56%	174,66	+17%	139,73	-7%	116,44	-22%	99,81	-33%
$\eta_v$	-	0,87	0,87	-1%	0,87	0%	0,87	0%	0,87	0%	0,87	0%
$p_{max}$	MPa	7,54	4,46	-41%	6,29	-17%	8,14	+8%	10,02	+33%	11,91	+58%
BMEP	MPa	0,96	0,77	-19%	0,90	-7%	0,99	+3%	1,06	+10%	1,11	+16%
$g_e$	g/kWh	287,87	355,32	+23%	307,61	+7%	280,34	-3%	262,36	-9%	249,42	-13%
$P_e$	kW	67,12	54,06	-19%	62,71	-7%	68,96	+3%	73,80	+10%	77,70	+16%
CO <sub>2</sub>	kg/h	61,06	60,70	-1%	60,95	0%	61,09	0%	61,18	0%	61,24	0%

Less than initial value

More than initial value

CR- compression ratio,  
 $V_c$  - clearance volume,  
 $\eta_v$  - volumetric efficiency,  
 $p_{max}$  - max pressure in chamber,  
BMEP - brake mean effective pressure,  
 $g_e$  - specific fuel consumption,  
 $P_e$  - effective power,  
CO<sub>2</sub> - carbon monoxide



DOWNSIZED ENGINE by 30% (for D=76,50mm and S=76,00mm)												
Variable Valve Timing in Intake support only												
SYMBOL	UNIT	INITIAL VALUE	RESULTS									
N1	-		1,10		1,25		1,40		1,55		1,70	
$\eta_v$	-	0,87	0,87	0%	0,87	0%	0,87	0%	0,87	0%	0,87	0%
$p_{max}$	MPa	7,54	7,13	-5%	7,40	-2%	7,80	+3%	8,41	+12%	9,36	+24%
BMEP	MPa	0,96	0,95	-1%	0,96	0%	0,97	+1%	1,01	+5%	1,10	+14%
$g_e$	g/kWh	287,87	290,53	1%	289,25	0%	284,32	-1%	273,13	-5%	251,93	-12%
$P_e$	kW	67,12	66,50	-1%	66,80	0%	67,96	+1%	70,74	+5%	76,69	+14%
CO <sub>2</sub>	kg/h	61,06	61,06	0%	61,06	0%	61,06	0%	61,06	0%	61,06	0%

Less than initial value

More than initial value

N1- polytrophic exponent for compression,  
 $\eta_v$  - volumetric efficiency,  
 $p_{max}$  - max pressure in chamber,  
BMEP - brake mean effective pressure,  
 $g_e$  - specific fuel consumption,  
 $P_e$  - effective power,  
CO<sub>2</sub> - carbon monoxide



DOWNSIZED ENGINE by 30% (for D=76,50mm and S=76,00mm)												
Variable Valve Timing in Exhaust support only												
SYMBOL	UNIT	INITIAL VALUE	RESULTS									
N2	-		1,10		1,25		1,40		1,55		1,70	
$\eta_v$	-	0,87	0,87	0%	0,87	0%	0,87	0%	0,87	0%	0,87	0%
$p_{max}$	MPa	7,54	7,54	0%	7,54	0%	7,54	0%	7,54	0%	7,54	0%
BMEP	MPa	0,96	1,16	+21%	0,92	-5%	0,72	-25%	0,56	-41%	0,43	-55%
$g_e$	g/kWh	287,87	238,85	-17%	301,75	+5%	383,34	+33%	491,49	+71%	639,28	+122%
$P_e$	kW	67,12	80,89	+21%	64,03	-5%	50,40	-25%	39,31	-41%	30,22	-55%
CO <sub>2</sub>	kg/h	61,06	61,06	0%	61,06	0%	61,06	0%	61,06	0%	61,06	0%


Less than initial value


More than initial value

N1- polytropic exponent for decompression,  
 $\eta_v$  - volumetric efficiency,  
 $p_{max}$  - max pressure in chamber,  
BMEP - brake mean effective pressure,  
 $g_e$  - specific fuel consumption,  
 $P_e$  - effective power,  
CO<sub>2</sub> - carbon monoxide



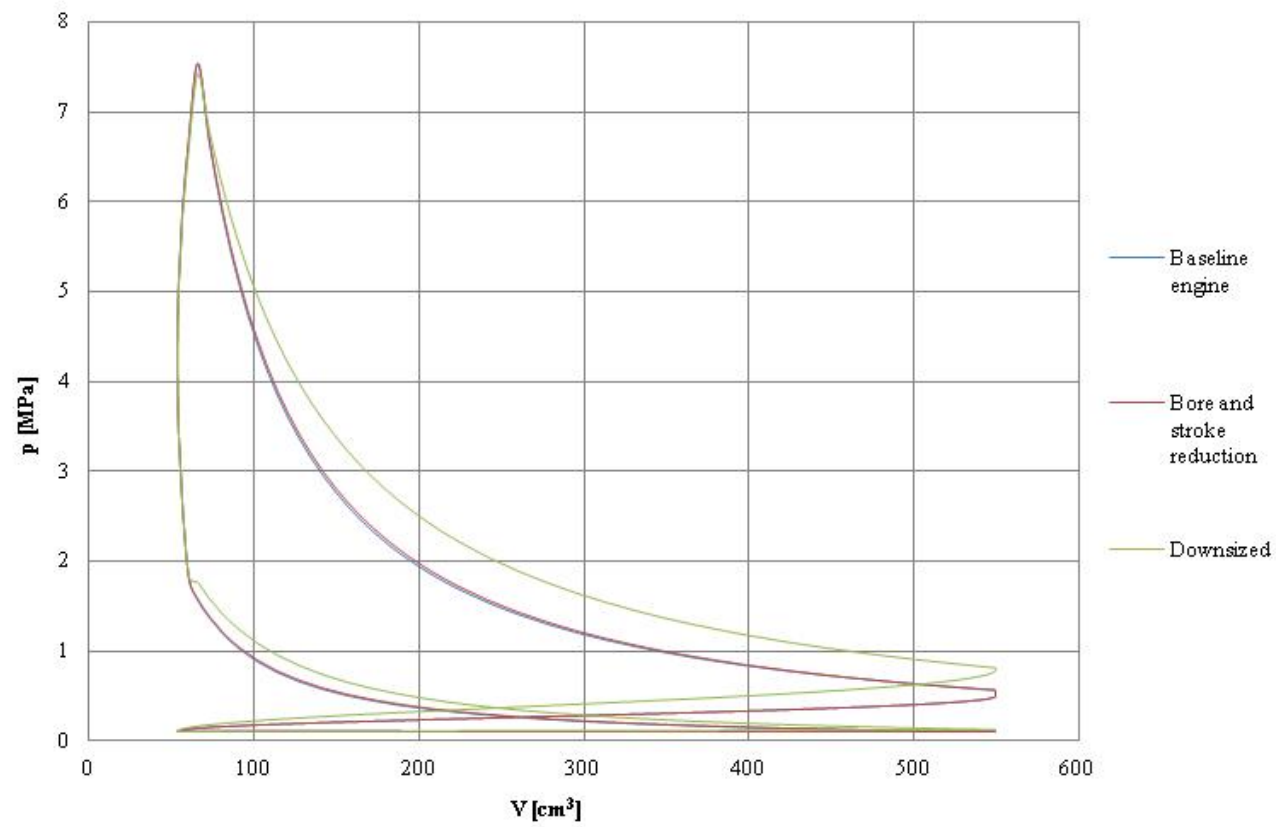
BASELINE ENGINE vs. DOWNSIZED ONE by 30%						
SYMBOL	UNIT	INITIAL VALUE	RESULTS			
			Downsized by geometry only		Downsized with all technologies	
D	mm	84,80	76,50	-10%	76,50	-10%
S	mm	88,00	76,00	-14%	76,00	-14%
V <sub>s</sub>	-	1988,04	1397,29	-30%	1397,29	-30%
η <sub>v</sub>	-	0,87	0,87	0%	1,19	+36%
p <sub>max</sub>	MPa	7,54	7,54	0%	7,42	-2%
BMEP	MPa	0,94	0,96	+3%	1,38	+47%
g <sub>e</sub>	g/kWh	295,10	287,87	-2%	273,69	-7%
P <sub>e</sub>	kW	93,16	67,12	-28%	96,26	+3%
CO <sub>2</sub>	kg/h	86,87	61,06	-30%	83,25	-4%

 Less than initial value

 More than initial value

D- cylinder diameter  
S- piston stroke  
V - swept volume,  
n<sub>v</sub> - volumetric efficiency,  
p<sub>max</sub> - max pressure in chamber,  
BMEP - brake mean effective pressure,  
g<sub>e</sub> - specific fuel consumption,  
P<sub>e</sub> - effective power,  
CO<sub>2</sub> - carbon monoxide







Study of effects of downsizing technologies shows possibility developing engine with reduced swept volume, with higher performance parameters and lower fuel consumption than in standard engine. Simple bore and stroke reduction causes reduction of engines performance but using modern technologies like charging, VCR and VVT allows to increase efficiency of combustion process but without synergy.

**Thank for your attention**

During goal realization, completely new, downsized engine was developed with 3% more power and fuel consumption decrease by 7% as well as giving possibility to CO<sub>2</sub> reduction by 4%.

